

# Sec 6.5

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## Sec. 6.5 Cryogenics and vacuum methods

- Requirement of detector: cooling for optimum performance
- Several categories of cooling system in cameras and spectrographs for ground-based :
  1. Thermoelectric coolers and liquid circulation coolers:
    - operate over the range  $-20\text{ }^{\circ}\text{C}$  to  $-50\text{ }^{\circ}\text{C}$
    - suitable for photomultiplier tubes, CCDs which have low dark current, and telescope-guiding cameras
    - popular in CCD cameras used on small telescopes because it is much easier and simpler to use
  2. Liquid and solid cryogenics
    - Dry ice (solid  $\text{CO}_2$ ):
      - cheap and readily available
      - Temperature around  $-78\text{ }^{\circ}\text{C}$  can be achieved with dry ice
    - Liquid nitrogen ( $\text{LN}_2$ ):
      - relatively cheap, cools detectors to  $-196\text{ }^{\circ}\text{C}$
      - used in almost all professional CCD cameras and near-infrared devices
    - Liquid helium ( $\text{LHe}$ ):
      - expensive
      - cool detectors to  $-269\text{ }^{\circ}\text{C}$
      - used for low-bandgap semiconductor materials and bolometers used in infrared instruments
      - usually pre-cooled with  $\text{LN}_2$  then blown out with  $\text{LN}_2$  gas and the  $\text{LHe}$  transfer begun

immediately

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- Cooling ability for liquid cryogenics:

- $\rho$ : density
- $L_V$ : latent heat of vaporization

$$(\rho L_V)_{\text{LHe}} = 0.74 \text{ W h L}^{-1}$$

$$(\rho L_V)_{\text{LN}_2} = 44.7 \text{ W h L}^{-1}$$

- Meaning:

- a 10 W heat load boils away 1 liter (L) of LHe in 0.074 hours
- a 10 W heat load boils away 1 liter (L) of LN<sub>2</sub> in 4.47 hours

- Liquid neon (LNe) and liquid hydrogen (LH):

- Intermediate solution between LHe and LN<sub>2</sub>
- rarely used in astronomy

### 3. Electrical heat engines or closed-cycle refrigerators:

- many infrared instruments and sub-millimeter/radio receivers employ multi-stage closed-cycle refrigerators (CCRs)
- vibration damping is needed, especially for CCR-cooled instruments in sensitive AO system
- counter-balance weights may also required

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### 4. $^3\text{He}$ systems:

- sub-millimeter and far-infrared bolometers
- temperature below 0.3 K are obtained by reducing the vapor pressure on the top of the liquid helium-3
- use internal sorption pump

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- Detector temperature affect on dark current and noise
- Steps in the thermal analysis of an instrument:
  - Determine what is to be actively cooled (ex. Detector only, whole instrument)
  - Tabulate the required operating temperature (ex. Detector, optics)
  - Calculate the thermal energy removed from the mass to be cooled
  - Determine the heat loads due to conduction and radiation
  - Select the appropriate cooling system
  - Estimate the cool-down time
  - Estimate the 'hold' time (for liquid cryogenics)
- Heat  $H$  (joules) removed from a mass  $m$  (kg) which is cooled from a temperature  $T_h$  to  $T_c$  is given by
  - $C$ : specific heat of the material
- The rate of transfer of heat  $Q_H$  (in watts) by conduction along a rod of uniform cross-sectional area ( $A$ ) and temperature gradient  $dT/dx$  is given by

$$Q_H = -kA \frac{dT}{dx}$$

- $k$ : thermal conductivity
- Thermal resistance:
  - $L$ : length of the conductor

$$R = \frac{1}{k} \frac{1}{A/L}$$

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- Since  $k$  is a function of temperature, it is often convenient to integrate over the required temperature range and give  $Q_H$  in the form:

$$Q_H = \frac{A}{L} [I_{T_h} - I_{T_c}]$$

- $L$ : length of conductor
- $I$ : tabulated property for many materials [watts/centimeter]

**Table 6.1.** Values of the thermal conductivity integrals in watts/centimeter for four materials.

| <i>Temperature<br/>(K)</i> | <i>OFHC Copper<br/>(watts/cm)</i> | <i>6061 Aluminum<br/>(watts/cm)</i> | <i>Stainless steel<br/>(watts/cm)</i> | <i>G-10 Fiberglass<br/>(watts/cm)</i> |
|----------------------------|-----------------------------------|-------------------------------------|---------------------------------------|---------------------------------------|
| 300                        | 1,520                             | 613                                 | 30.6                                  | 1.00                                  |
| 250                        | 1,320                             | 513                                 | 23.4                                  | 0.78                                  |
| 200                        | 1,120                             | 413                                 | 16.6                                  | 0.55                                  |
| 150                        | 915                               | 313                                 | 10.6                                  | 0.37                                  |
| 100                        | 700                               | 211                                 | 6.3                                   | 0.19                                  |
| 77                         | 586                               | 158                                 | 3.2                                   | 0.11                                  |
| 50                         | 426                               | 89.5                                | 1.4                                   | 0.07                                  |
| 10                         | 25                                | 3.6                                 | 0.03                                  | 0.005                                 |

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- Power radiated from a body of area  $A$  at an absolute temperature  $T$  is given by:

$$Q_H = \varepsilon \sigma A T^4$$

- Stefan-Boltzmann constant  $\sigma = 5.67 \times 10^{-8} \text{ W m}^{-2} \text{ K}^{-4}$
- $\varepsilon$ : emissivity of the surface
- The net rate of heat transfer by radiation from a body temperature  $T_h$  onto a body temperature  $T_c$  is given by:

$$Q_H = \sigma A_c F_{hc} [T_h^4 - T_c^4]$$

- $F_{hc}$ : effective emissivity which depends on the geometry of the cryostat configuration
- The cooling time for a mass  $m$  from initial temperature  $T_i$  to  $T_{i+1}$ :

$$\Delta t_{T_i, T_{i+1}} = \frac{m C_p(T_i) [T_i - T_{i+1}]}{(A/L) k(T_i) [T_i - T_{\text{sink}}] - Q(T_i)}$$

- $T_{\text{sink}}$ : temperature of the cold source (such as  $\text{LN}_2$ )
- $Q(T)$ : rate at which heat comes back into the mass at temperature  $T_i$

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- The larger the surface area of a cryostat, the greater the deformation due to the atmospheric pressure
  - The deflection at the center of a flat end plate clamped at its edge:

$$\delta = \frac{3(1 - \mu^2)R^4}{16Ed^3} P$$

- The maximum tensile stress, which occurs at the edge, is given by:

$$S_{\max} = \frac{3}{4} \left( \frac{R}{d} \right)^2 P$$

- $\mu$ : Poisson's ratio (typically 0.3 for metals)
  - E: elasticity or Young's modulus
  - R: radius of the plate
  - d: thickness
  - P: external pressure
- The deflection of a circular plate with unclamped edges, and maximum tensile stress at the center:

$$\delta = \frac{3(1 - \mu)(5 + \mu)R^4}{16Ed^3} P$$

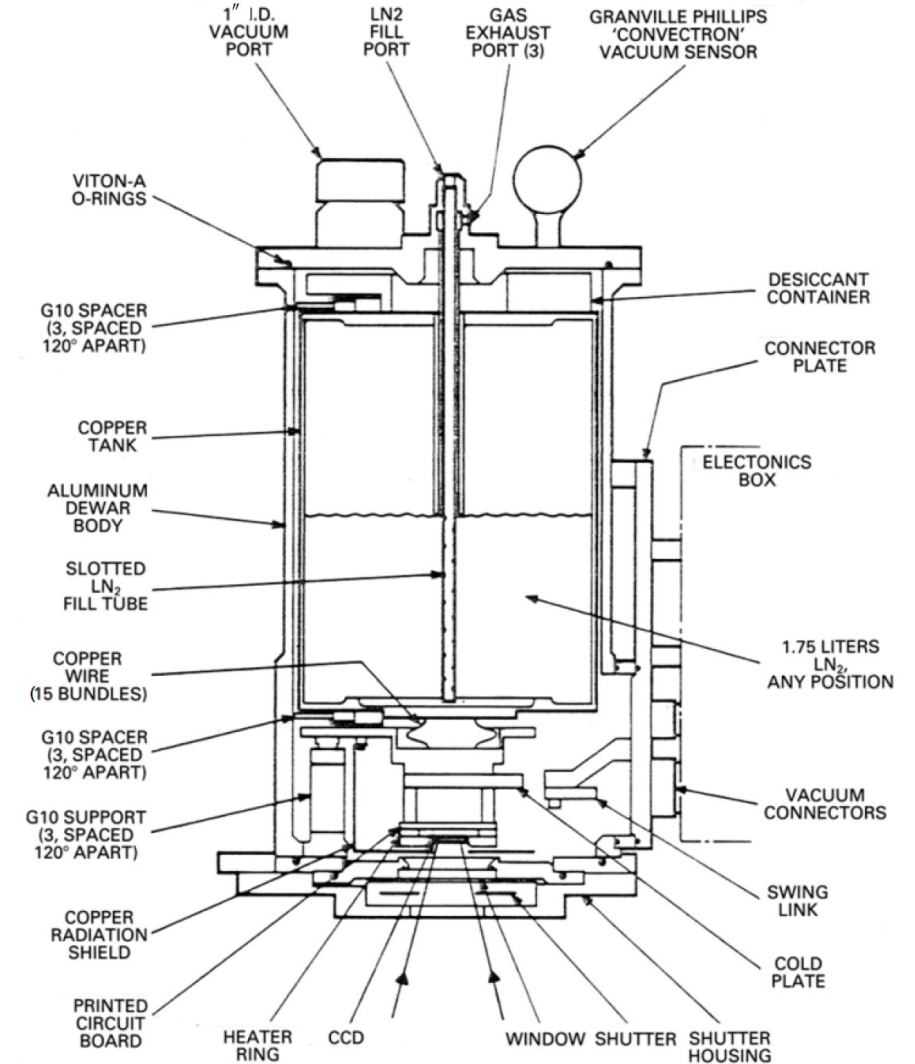
$$S_{\max} = \frac{3}{8} \left( \frac{R}{d} \right)^2 (3 + \mu) P$$

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- Design objective for a good cooling system should include the followings:
  1. Minimum detector movement
  2. Minimum effort to keep the system uniformly cold
  3. Good accessibility inside the cryostat
  4. Cost and manufacturing time
- A typical LN<sub>2</sub> cryostat used with a CCD is shown schematically in Figure 6.11
- Mean free path, average distance traveled between molecules collisions:

$$\lambda_{\text{mfp}} = \frac{1}{\sqrt{2}n\pi d^2}$$

- n: number density of molecules
- d: diameter of the molecule



**Figure 6.11.** A cross-sectional view of a typical liquid-nitrogen (LN<sub>2</sub>) cryostat illustrating all of the components needed in its construction. Credit: NOAO.

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- Net pumping speed of a pump:

$$\frac{1}{S} = \frac{1}{S_{\text{pump}}} + \frac{1}{C_{\text{lines}}}$$

- S: pumping speed
- C: conductance which depends on gas pressure and viscosity
  - When mean free path is small:

$$C = 180 \frac{D^4}{L} P_{\text{av}}$$

- When mean free path is large:

$$C = 12 \frac{D^3}{L}$$

- D: diameter of circular tube
    - L: length [centimeters]
    - $P_{\text{av}}$ : pressure

- Pump down time (in seconds):

$$t = 2.3 \frac{V}{S} \ln \left( \frac{P_0}{P} \right)$$

- Time required to collect refrigerant into the receiver or compressor and create a vacuum in the low-pressure side of the system
- V: volume, S: net pumping speed,  $P_0$ : initial pressure, P: final pressure